MATHEMATICAL MODELLING AND ANALYSIS VOLUME 10 NUMBER 2, 2005, PAGES 173–190 © 2005 Technika ISSN 1392-6292

# INCREASING OF ACCURACY FOR ENGINEERING CALCULATION OF HEAT TRANSFER PROBLEMS IN TWO LAYER MEDIA

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Received September 21, 2004; revised March 12, 2005

**Abstract.** In this paper we study the simple algorithms for modelling the heat transfer problem in two layer media. The initial model which is based on a partial differential equation is reduced to ordinary differential equations (ODEs). The increase of accuracy is shown if instead of first order ODE initial value problem ([4, 5]) the second order differential equations is taken. Such a procedure allows us to obtain a simple engineering algorithm for solving heat transfer equations in two layered domain of Cartesian, cylindrical (with axial symmetry) and spherical coordinates (with radial symmetry). In a stationary case the exact finite difference scheme is obtained.

Key words: heat transfer, layered media, numerical methods, mathematical modelling

### 1. The Mathematical Model

We shall consider the partial differential equations [4]:

$$\gamma_k \frac{\partial u_k}{\partial t} = \frac{1}{p(x)} \frac{\partial}{\partial x} \left( \lambda_k p(x) \frac{\partial u_k}{\partial x} \right) - q_k(x, t) \tag{1.1}$$

in multilayered domain  $\Omega$  with N layers

$$\Omega = \{x : x \in [x_{k-1}, x_k], k = \overline{1, N}\}$$

where  $x_0 = 0$ ,  $x_N = L$ ,  $h_k = x_k - x_{k-1}$ ,  $u_k = u_k(x, t)$  is the absolute temperature [K] in the layer  $[x_{k-1}, x_k]$ ,  $\gamma_k = c_k \rho_k$ ,  $c_k [\frac{J}{kg.K}]$ ,  $\rho_k [\frac{kg}{m^3}]$ ,  $\lambda_k [\frac{W}{m.K}]$  are corresponding constants of specific thermal capacity, density and coefficients of heat

conductivity in every layer, t[s] is the time,  $q_k = q_k(x, t)$  is the function of thermal sources, x[m] is the space coordinate, p = p(x) is given function depending on the system of coordinates: p = 1 in the Cartesian coordinates, p = x in cylindrical coordinates with an axial symmetry,  $p = x^2$  in spherical coordinates with a radial symmetry.

Adding continuity conditions on surfaces  $x = x_k, k = \overline{1, N-1}$ 

$$\begin{cases} u_k(x_k,t) = u_{k+1}(x_k,t) \\ \lambda_k \frac{\partial u_k(x_k,t)}{\partial x} = \lambda_{k+1} \frac{\partial u_{k+1}(x_k,t)}{\partial x}, \end{cases}$$
(1.2)

boundary conditions on the surfaces  $x = x_0 = 0$ ,  $x = x_n = L$ 

$$\begin{cases} \lambda_1 p(0) \frac{\partial u_1(0,t)}{\partial x} = \alpha_0 (u_1(0,t) - T_0) \\ \lambda_N p(L) \frac{\partial u_N(L,t)}{\partial x} = f(u_N(L,t)) \end{cases}$$
(1.3)

and the initial condition at t = 0

$$u_k(x,t) = \phi(x), k = \overline{1,N}$$
(1.4)

we obtain the initial-boundary value problem (1.1-1.4) for the heat transfer equation. The nonlinear function  $f(u_N)$  in the boundary condition (1.3) describes the radiation from heaters and convection, for example

$$f(u_N(L,t)) = \alpha_L(T_L - u_N(L,t)) + \epsilon \sigma(T_*^4 - u_N^4(L,t)),$$

where  $\alpha_0, \alpha_L[\frac{W}{m^3.K}]$  are the coefficients of heat transfer,  $\epsilon$  is the coefficient of emissivity ( $\epsilon \in [0,1]$ ),  $\sigma = 5.6703.10^{-8}[\frac{W}{m^2.K^4}]$  is the Stefan-Boltzmann constant,  $T_0, T_L, T_*$  are the constants of the temperatures outside the media and outside the heaters,  $\phi = \phi(x)$  is the given initial temperature.

If  $\alpha_0 = \alpha_L = \infty$ , then we have the first kind boundary conditions in the form

$$u_1(0,t) = T_0, \quad u_N(L,t) = T_L.$$

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If p(0) = 0 then the first boundary condition (1.3) is omitted and we can consider the symmetry condition

$$\frac{\partial u_1(0,t)}{\partial x} = 0. \tag{1.5}$$

In the case of homogeneous media we consider the following partial differential equation

$$\frac{\partial u}{\partial t} = \frac{1}{p(x)} \frac{\partial}{\partial x} \left( p(x) \frac{\partial u}{\partial x} \right) - q(x, t), \tag{1.6}$$

where the constants of heat transfer parameters  $c, \rho, \lambda$  are normalizing magnitudes and  $c\rho/\lambda$  is used as appropriate factor to the time t and function q.

In every layer the heat equation (1.1) can be presented in the following form

$$\frac{\partial}{\partial x} \left( \lambda_k \frac{\partial u_k(x,t)}{\partial x} \right) = F_k, \quad k = \overline{1,N}, \tag{1.7}$$

where  $F_k = \gamma_k \dot{u}_k + q_k, \ \dot{u}_k = \frac{\partial u_k}{\partial t}.$ 

# 2. The Exact 3-Points Finite-Difference Scheme

We use the method of finite volumes [3] for approximation of the differential problem. We consider N + 1 grid points in the x- direction

$$0 = x_0 < x_1 < \ldots < x_N = L.$$

Then the exact fi nite-difference scheme for a given function  $F_k$  is defined in the form [2]

$$\lambda_1 a_1 (u_1 - u_0) - \alpha_0 (u_0 - T_0) = \bar{R}_0^+, \qquad (2.1)$$

$$\lambda_{k+1}a_{k+1}(u_{k+1} - u_k) - \lambda_k a_k(u_k - u_{k-1}) = \bar{R}_k, \ k = \overline{1, N-1},$$
(2.2)

$$f(u_N) - \lambda_N a_N (u_N - u_{N-1}) = \bar{R}_N^-,$$
(2.3)

where

$$\begin{split} \bar{R}_{k} &= \bar{R}_{k}^{+} + \bar{R}_{k}^{-}, \quad \bar{R}_{k}^{+} = I_{k}^{+} + \gamma_{k+1} R_{k}^{+}, \quad \bar{R}_{k}^{-} = I_{k}^{-} + \gamma_{k} R_{k}^{-}, \\ I_{k} &= I_{k}^{+} + I_{k}^{-}, \quad u_{k} = u_{k}(t) = u_{k}(x_{k}, t), \quad k = \overline{1, N}, \quad u_{0} = u_{1}(0, t), \\ R_{k}^{-} &= \int_{x_{k-1}}^{x_{k}} \left(1 - a_{k} \int_{x}^{x_{k}} \frac{d\psi}{p(\psi)}\right) p(x) \dot{u}_{k}(x, t) dx \\ &= a_{k} \int_{x_{k-1}}^{x_{k}} \left(p(x) \dot{u}_{k}(x, t) \int_{x_{k-1}}^{x} \frac{d\psi}{p(\psi)}\right) dx, \\ R_{k}^{+} &= \int_{x_{k}}^{x_{k+1}} \left(1 - a_{k+1} \int_{x_{k}}^{x} \frac{d\psi}{p(\psi)}\right) p(x) \dot{u}_{k+1}(x, t) dx \\ &= a_{k+1} \int_{x_{k}}^{x_{k+1}} \left(p(x) \dot{u}_{k+1}(x, t) \int_{x}^{x_{k+1}} \frac{d\psi}{p(\psi)}\right) dx, \\ a_{k} &= \frac{1}{\int_{x_{k-1}}^{x_{k}} \frac{dx}{p(x)}, \quad I_{k}^{+} = a_{k+1} \int_{x_{k}}^{x_{k+1}} \left(p(x) q_{k}(x, t) \int_{x}^{x_{k+1}} \frac{d\psi}{p(\psi)}\right) dx, \\ I_{k}^{-} &= \int_{x_{k-1}}^{x_{k}} \left(1 - a_{k} \int_{x}^{x_{k}} \frac{d\psi}{p(\psi)}\right) p(x) q_{k}(x, t) dx \\ &= a_{k} \int_{x_{k-1}}^{x_{k}} \left(p(x) q_{k}(x, t) \int_{x_{k-1}}^{x} \frac{d\psi}{p(\psi)}\right) dx. \end{split}$$

If p(0) = 0 (cylindrical and spherical coordinates), then  $a_1 = \alpha_0 = 0$  and the difference equation (2.1) is defined as

$$\lambda_1(u_1 - u_0) = R_0^*, \tag{2.4}$$

and the equation (2.2) for k = 1 is given as

$$\lambda_2 a_2 (u_2 - u_1) = R_1^*, \tag{2.5}$$

where

$$R_0^* = \gamma_1 \int_0^{x_1} p(x) \dot{u}_1(x,t) \int_x^{x_1} \frac{d\psi}{p(\psi)} dx + \int_0^{x_1} p(x) q_1(x,t) \int_x^{x_1} \frac{d\psi}{p(\psi)} dx,$$
  

$$R_1^* = \bar{R}_1^+ + \gamma_1 \int_0^{x_1} p(x) \dot{u}_1(x,t) dx + \int_0^{x_1} p(x) q_1(x,t) dx.$$

Summarizing expressions (2.2), (2.3) we obtain equation

$$f(u_N) = Q, \quad Q = \sum_{k=2}^{N-1} \bar{R}_k + \bar{R}_N^- + R_0^* + R_1^*.$$

This equation has a positive root  $u_N > 0$ , if  $\epsilon \sigma T_*^4 + \alpha_L T_L > Q$ .

Next we consider only two layers, then  $N = 2, x_1 = h_1, x_2 = L = h_1 + h_2$ . Then the unknown values are  $u_1, u_2(p(0) = 1, \alpha_0 = \infty)$  or  $u_0, u_1, u_2(p(0) = 0)$ .

# 3. The Cartesian Coordinates



Figure 1. The calculated temperature  $u_1$  in the layer  $x = x_1$  in the case of Cartesian coordinates.

In this case (see fig. 1)

$$p(x) = 1, \quad a_k = \frac{1}{h_k}, \quad R_k^- = \frac{1}{h_k} \int_{x_{k-1}}^{x_k} (x - x_{k-1}) \dot{u}_k(x, t) \, dx,$$
$$R_k^+ = \frac{1}{h_{k+1}} \int_{x_k}^{x_{k+1}} (x_{k+1} - x) \dot{u}_{k+1}(x, t) \, dx, \quad I_k^- = \frac{1}{h_k} \int_{x_{k-1}}^{x_k} (x - x_{k-1}) q_k \, dx$$
$$I_k^+ = \frac{1}{h_{k+1}} \int_{x_k}^{x_{k+1}} (x_{k+1} - x) q_{k+1} \, dx,$$

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and the finite-difference scheme ( $\alpha_0 = \infty$ ) is given in the form

$$\begin{cases} \frac{\lambda_2}{h_2}(u_2 - u_1) - \frac{\lambda_1}{h_1}(u_1 - T_0) = \gamma_2 R_1^+ + \gamma_1 R_1^- + I_1 \\ f(u_2) - \frac{\lambda_2}{h_2}(u_2 - u_1) = \gamma_2 R_2^- + I_2^-. \end{cases}$$
(3.1)

In the stationary case, if

$$q_k = const, \ \dot{u}_k = 0, \ I_1 = 0.5(q_1h_1 + q_2h_2), \ I_2 = 0.5q_2h_2$$

then

$$f(u_2) + D_{12}((T_0 - u_2)D_1 - Q_{12}) = Q_2, \quad u_1 = \frac{D_2u_2 + D_1T_0 - Q_{12}}{D_1 + D_2}, \quad (3.2)$$

where

$$D_i = \frac{\lambda_i}{h_i}, \ Q_i = 0.5q_ih_i, \ i = 1, 2, \ D_{12} = \frac{D_2}{D_1 + D_2}, \ Q_{12} = Q_1 + Q_2.$$

Therefore the value  $u_2$  can be obtained from the equation  $u_2^3 + a = \frac{b}{u_2}$ , where

$$a = \frac{\alpha_L + D_{12}D_1}{\epsilon\sigma} > 0, \ b = \frac{\epsilon\sigma T_*^4 + \alpha_L T_L + D_{12}D_1 T_0 - Q_2 - D_{12}Q_{12}}{\epsilon\sigma}$$

This equation have a positive root for b > 0, e.g.,  $q_1 \le 0, q_2 \le 0$ . The exact solution of the stationary problem (1.1)–(1.3) satisfi es expressions (3.2).

In the non-stationary case  $(\dot{u}_k \neq 0)$ , initial-value problems for ODE are used and we compute integrals  $R_i^{\pm}$  approximately by the following quadrature formulas

$$J_m = A_1^{(m)} V_2(0) + A_2^{(m)} V_2(1) + A_3^{(m)} V_2'(1) + B_2^{(m)} V_2''(1) + r_m, \ m = 1, 2, \ (3.3)$$
  
$$J_3 = A_1^{(3)} V_1(0) + A_2^{(3)} V_1(1) + A_3^{(3)} V_1'(0) + B_1^{(3)} V_1''(0) + r_3, \ (3.4)$$

where

$$J_{1} = \frac{R_{1}^{+}}{h_{2}} = \int_{0}^{1} (1 - \bar{x}) V_{2}(\bar{x}) d\bar{x}, \quad J_{2} = \frac{R_{2}^{-}}{h_{2}} = \int_{0}^{1} \bar{x} V_{2}(\bar{x}) d\bar{x},$$
$$J_{3} = \frac{R_{1}^{-}}{h_{1}} = \int_{0}^{1} \bar{x} V_{1}(\bar{x}) d\bar{x}, \quad V_{1}(\bar{x}) = \dot{u}_{1}(\bar{x}h_{1}, t), \quad V_{2}(\bar{x}) = \dot{u}_{2}(h_{1} + \bar{x}h_{2}, t),$$

 $r_{m} = \frac{h_{2}^{4}}{4!} \frac{\partial^{4} \dot{u}(\xi_{m}, t)}{\partial x^{4}} C_{m}, \quad \xi_{m} \in (0, L), (m = 1, 2, 3) \text{ are the errors terms,} \\ A_{j}^{(m)}, B_{j}^{(m)}, C_{m}, (j, m = 1, 2, 3) \text{ are the indefi nite coefficients.} \\ \text{Using the power functions } V_{j}(\bar{x}) = \bar{x}^{i}, i = \overline{0, 4}, j = 1, 2 \text{ we get the systems of}$ 

linear algebraic equations for  $A_j^{(m)}, B_j^{(m)}$ :

$$\begin{cases} g(i,m) = A_1^{(m)}0^i + A_2^{(m)} + iA_3^{(m)} + i(i-1)B_2^{(m)}, \\ \frac{1}{i+2} = A_1^{(3)}0^i + A_2^{(3)} + iA_3^{(3)}0^{i-1} + i(i-1)B_1^{(3)}0^{i-2}, \end{cases}$$
(3.5)

where

$$m = 1, 2, \ 0^0 = 1, \ g(i, 1) = \frac{1}{(i+1)(i+2)}, \ g(i, 2) = \frac{1}{i+2}.$$

The solutions of these systems are given as

$$\begin{aligned} A_1^{(1)} &= \frac{1}{5}, \quad A_2^{(1)} &= \frac{3}{10}, \quad A_3^{(1)} &= -\frac{2}{15}, \quad B_2^{(1)} &= \frac{1}{40}, \\ A_1^{(2)} &= \frac{1}{20}, \quad A_2^{(2)} &= \frac{9}{20}, \quad A_3^{(2)} &= -\frac{7}{60}, \quad B_2^{(2)} &= \frac{1}{60}, \\ A_1^{(3)} &= \frac{3}{10}, \quad A_2^{(3)} &= \frac{1}{5}, \quad A_3^{(3)} &= \frac{2}{15}, \quad {}_1^{(3)} &= -\frac{1}{40}. \end{aligned}$$

Constants  $C_m$  in the residual  $r_m$  are determined from (3.3) – (3.4), for  $V_1(\bar{x}) = V_2(\bar{x}) = \bar{x}^4$ :  $C_1 = C_3 = -\frac{1}{30}, C_2 = -\frac{1}{60}.$ 

Using the difference equations (3.1) and the right hand side integrals approximating by (3.3), (3.4) with neglected error terms  $r_m, m = 1, 2, 3$ , the approximate numerical solution  $u_1(t), u_2(t)$  at every time step t > 0 can be found by solving the following stiff system of two nonlinear ODEs of the second order  $(\dot{u}_0 = \ddot{u}_0 = 0, \ddot{u} = \frac{\partial^2}{\partial t^2}, \alpha_0 = \infty)$ :

$$\begin{cases} \gamma_{2}h_{2}\left[A_{1}^{(1)}\dot{u}_{1}+A_{2}^{(1)}\dot{u}_{2}+\frac{h_{2}}{\lambda_{2}}\left(A_{3}^{(1)}f_{u}'(u_{2})\dot{u}_{2}+h_{2}B_{2}^{(1)}(\dot{q}_{2}(L,t)+\gamma_{2}\ddot{u}_{2})\right)\right]\\ +\gamma_{1}h_{1}\left[A_{2}^{(3)}\dot{u}_{1}+\frac{h_{1}^{2}}{\lambda_{1}}B_{1}^{(3)}\dot{q}_{1}(0,t)\right]+I_{1}=\frac{\lambda_{2}}{h_{2}}(u_{2}-u_{1})-\frac{\lambda_{1}}{h_{1}}(u_{1}-T_{0}), \end{cases}$$

$$(3.6)$$

$$\begin{cases} \gamma_2 h_2 \Big[ A_1^{(2)} \dot{u}_1 + A_2^{(2)} \dot{u}_2 + \frac{h_2}{\lambda_2} \Big( A_3^{(2)} f_u'(u_2) \dot{u}_2 \\ + h_2 B_2^{(2)} (\dot{q}_2(L,t) + \gamma_2 \ddot{u}_2) \Big) \Big] + I_2^- = f(u_2) - \frac{\lambda_2}{h_2} (u_2 - u_1). \end{cases}$$
(3.7)

Here one should take into account that from (1.1), (1.3) it follow that

$$\begin{split} V_2'(1) &= h_2 \frac{\partial}{\partial x} \dot{u}_2(L,t) = h_2 \frac{\partial}{\partial t} u_2'(L,t) = \frac{h_2}{\lambda_2} \dot{f}(u_2) = \frac{h_2}{\lambda_2} f_u'(u_2) \dot{u}_2, \\ f_u'(u_2) &= -(\alpha_L + 4\epsilon \sigma u_2^3), \quad V_1'(0) = h_1 \frac{\partial}{\partial x} \dot{u}_1(0,t) = h_1 \frac{\partial}{\partial t} u_1'(0,t) = h_1 \alpha_0 \dot{u}_0, \\ V_2''(1) &= h_2^2 \frac{\partial^2}{\partial x^2} \dot{u}(L,t) = h_2^2 \frac{\partial}{\partial t} u''(L,t) = \frac{h_2^2}{\lambda_2} \frac{\partial}{\partial t} (\gamma_2 \dot{u}(L,t) + q(L,t)) \\ &= \frac{h_2^2}{\lambda_2} (\gamma_2 \ddot{u}_2 + \dot{q}_2). \end{split}$$

The initial conditions for ODEs (3.6), (3.7) are the following

$$\begin{cases} u_1(0) = \phi(h_1), \ u_2(0) = \phi(L), \\ \dot{u}_1(0) = (\lambda_1 \phi''(h_1) - q_1(h_1, 0))/\gamma_1, \\ \dot{u}_2(0) = (\lambda_2 \phi''(L) - q_2(L, 0))/\gamma_2. \end{cases}$$
(3.8)

If the derivatives  $\ddot{u}$  are not used in (3.3), (3.4) then [3]

$$\begin{cases} R_{k}^{-} = \gamma_{k}h_{k} \Big( \frac{1}{6} \dot{u}_{k-1} + \frac{1}{3} \dot{u}_{k} - \frac{h_{k}^{2}}{24} \frac{\partial^{2} \dot{u}(\xi_{k}^{-}, t)}{\partial x^{2}} \Big), \\ R_{k}^{+} = \gamma_{k+1}h_{k+1} \Big( \frac{1}{3} \dot{u}_{k} + \frac{1}{6} \dot{u}_{k+1} - \frac{h_{k+1}^{2}}{24} \frac{\partial^{2} \dot{u}(\xi_{k}^{+}, t)}{\partial x^{2}} \Big), \end{cases}$$
(3.9)

where  $\xi_k^- \in (x_{k-1}, x_k), \, \xi_k^+ \in (x_k, x_{k+1}).$ 

For the equation (1.6),  $h_1 = h_2 = h$  the finite difference scheme (3.1) is given in the form

$$\begin{cases} u_2 - 2u_1 + T_0 = h(R_1 + I_1) \\ hf(u_2) - u_2 + u_1 = h(R_2^- + I_2^-), \end{cases}$$
(3.10)

where

$$\frac{R_1}{h} = J_1 = \int_0^1 \bar{x} V(\bar{x}) \, d\bar{x} + \int_1^2 (2 - \bar{x}) V(\bar{x}) d\bar{x}, 
\frac{R_2^-}{h} = J_2 = \int_1^2 (\bar{x} - 1) V(\bar{x}) \, d\bar{x}, \quad V(\bar{x}) = \dot{u}(\bar{x}h, t), \bar{x} = x/h, 
\begin{cases} J_m = A_1^{(m)} V(0) + A_2^{(m)} V(1) + A_3^{(m)} V(2) + A_4^{(m)} V'(2) \\ + B_1^{(m)} V''(0) + B_2^{(m)} V''(1) + B_3^{(m)} V''(2) + r_m, \end{cases}$$
(3.11)

 $r_m = \frac{h^7}{7!} \frac{\partial^7 \dot{u}(\xi_m, t)}{\partial x^7} C_m, \, \xi_m \in (0, L), \, m = 1, 2. \text{ Using power functions } V(\bar{x}) = \bar{x}^i, i = 0, \dots, 6 \text{ in the expressions (3.11) we obtain two systems of 7 linear algebraic equations for determination of <math>A_j^{(m)}, B_j^{(m)}$ 

$$g(i,m) = A_1^{(m)}0^i + A_2^{(m)} + A_3^{(m)}2^i + A_4^{(m)}i2^{i-1} + i(i-1)(B_1^{(m)}0^{i-2} + B_2^{(m)} + B_3^{(m)}2^{i-2}), \quad m = 1, 2, \quad (3.12)$$

where

$$g(i,1) = \frac{2^{i+2}-2}{(i+1)(i+2)}, \quad g(i,2) = \frac{i2^{i+1}+1}{(i+1)(i+2)}$$

Constants  $C_m$  are determined from (3.11) for  $V(\bar{x}) = \bar{x}^n, n = 7$ :

$$C_m = g(n,m) - A_2^{(m)} - A_3^{(m)} 2^n - A_4^{(m)} n 2^{n-1} - n(n-1)(B_2^{(m)} + B_3^{(m)} 2^{n-2}).$$
(3.13)

The solution of system (3.12) is given by:

$$\begin{split} A_1^{(1)} &= A_3^{(1)} = \frac{11}{252}, \quad A_2^{(1)} = \frac{115}{126}, \quad A_4^{(1)} = 0, \quad B_2^{(1)} = \frac{313}{7560}, \\ B_1^{(1)} &= B_3^{(1)} = -\frac{13}{15120}, \quad A_1^{(2)} = A_3^{(2)} = \frac{109}{504}, \quad A_2^{(2)} = \frac{269}{252}, \\ A_4^{(2)} &= 0, \quad B_1^{(2)} = B_3^{(2)} = -\frac{83}{30240}, \quad B_2^{(2)} = \frac{1223}{15120}. \end{split}$$

We have from (3.13) that  $C_1 = 0, C_2 = \frac{32}{9}$ . Therefore we get the residuals in the form  $r_1 = \frac{h^8}{8!} \frac{\partial^8 \dot{u}(\xi_1, t)}{\partial x^8} C_1$ , and for n = 1 we have from (3.13)  $C_1 = \frac{59}{1890}$ . If  $B_3^{(1)} = B_1^{(1)} = 0$ , then

$$A_1^{(1)} = A_3^{(1)} = \frac{1}{30}, \ A_2^{(1)} = \frac{28}{30}, \ B_2^{(1)} = \frac{1}{20}, \ r_1 = -\frac{1}{420} \frac{\partial^6 \dot{u}(\xi, t)}{\partial x^6} h^6, \ \xi \in (0, L).$$

The system of ODEs (3.6)–(3.7) is presented in the form

$$h^{2}(A_{2}^{(1)}\dot{u}_{1} + A_{3}^{(1)}\dot{u}_{2}) + h^{4}(B_{1}^{(1)}\dot{q}_{0} + B_{2}^{(1)}(\dot{q}_{1} + \ddot{u}_{1}) + B_{3}^{(1)}(\dot{q}_{2} + \ddot{u}_{2})) + I_{1} = u_{2} - 2u_{1} + T_{0}, \qquad (3.14)$$
$$h^{2}(A_{2}^{(2)}\dot{u}_{1} + A_{3}^{(2)}\dot{u}_{2}) + h^{4}(B_{1}^{(2)}\dot{q}_{0} + B_{2}^{(2)}(\dot{q}_{1} + \ddot{u}_{1})$$

$$+ B_3^{(2)}(\dot{q}_2 + \ddot{u}_2)) + I_2^- = hf(u_2) - u_2 + u_1, \quad (3.15)$$

where  $\dot{q}_i = \dot{q}(x_i, t), j = 0, 1, 2$ . If the derivatives are not used in (3.11) then [3]

$$A_1^{(1)} = A_3^{(1)} = \frac{1}{12}, \ A_2^{(1)} = \frac{10}{12}, \ r_1 = \frac{h^4}{4!} \frac{\partial^4 \dot{u}(\xi, t)}{\partial x^4} C_1, \ C_1 = -\frac{1}{10}.$$

Example 1. If

$$q = 0, \ \alpha_L = \infty, \ \phi(x) = \sin(\frac{\pi x}{L}), \ \dot{u}_2 = \ddot{u}_2 = 0, \ T_0 = T_L = 0,$$

then from the first order ODE (3.14) with  $A_2^{(1)} = \frac{5}{6}, B_2^{(1)} = 0, u_1(0) = 1$  it follows that  $u_1(t) = \exp(-9.6t/L^2)$ . The exact solution of (1.6) is  $u(x,t) = \exp(-\pi^2 t/L^2) \sin(\frac{\pi x}{L})$  or  $u_1 = u(h,t) = \exp(-\pi^2 t/L)$ . Using the approximation

$$u''(h,t) \approx \Lambda u_1 = \frac{1}{h^2} \big( u_0(t) - 2u_1(t) + u_2(t) \big),$$

we get thefi rst order ODE (the method of lines)  $\dot{u}_1 = \exp(-8t/L^2)$ . The second order ODE (3.14) is given as

$$b_1\ddot{u}_1 + a_1\dot{u}_1 + u_1 = 0 \ (b_1 = h^4 B_2^{(1)}, a_1 = h^2 A_2^{(1)})$$

with initial conditions  $u_1(0) = 1$ ,  $\dot{u}_1(0) = \phi''(h) = -(\pi/L)^2$ . It's solution is given by

$$u_1(t) = D_1 \exp(\mu_1 t) + D_2 \exp(\mu_2 t)$$

where  $\mu_{1,2} = -a_1/(2b_1) \pm \sqrt{(a_1/(2b_1))^2 - 1/b_1}, D_1 = (\mu_2 + (\pi/L)^2)/(\mu_2 - \mu_1), D_2 = -((\pi/L)^2 + \mu_1)/(\mu_2 - \mu_1).$ 

Using the approximation  $u''(h,t) \approx \Lambda u_1 - \frac{h^2}{12} u^{(4)}(h,t), u^{(4)}(h,t) = \ddot{u}_1(t)$ , we have the second order ODE with  $b_1 = \frac{h^4}{24}$ ,  $a_1 = \frac{h^2}{2}$  (the method of lines with a high

order approximation).

The results of calculations are presented in Table 1. Here L = 2,  $u_{1*}$  is the analytical solution,  $u_{1pp}$  is a  $O(h^8)$  order approximation,  $u_{1p}$  is  $O(h^6)$  order approximation,  $u_1$  is a usual approximation,  $u_{1t}$  is obtained by using the method of lines,  $u_{1tt}$  is computed by the method of lines of high approximation.

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**Table 1.** The values u(1, t) at different time t moments.

t	$u_{1*}$	$u_{1pp}$	$u_{1p}$	$u_1$	$u_{1tt}$	$u_{1t}$
.1	.781344	.781340	.78127	.7866	.770	.82
.2	.610498	.610490	.61032	.6188	.607	.67
.5	.291213	.291201	.29094	.3012	.284	.37
.9	.108537	.108529	.10834	.1153	.103	.16

# 4. The Cylindrical Coordinates



**Figure 2.** The calculated temperatures  $u_0$ ,  $u_1$  in layers  $x_0 = 0$  and  $x = x_1$  in the case of cylindrical coordinates.

In the cylindrical coordinates (see fi g.2) we get the following coeffi cients:

$$p(x) = x, \quad a_k = \frac{1}{\ln \frac{x_k}{x_{k-1}}}, \ k \ge 2, \quad a_1 = 0,$$
  
$$R_k^- = a_k \int_{x_{k-1}}^{x_k} x \ln \frac{x}{x_{k-1}} \dot{u}_k(x,t) \, dx, \quad R_k^+ = a_{k+1} \int_{x_k}^{x_{k+1}} x \ln \frac{x_{k+1}}{x} \dot{u}_{k+1}(x,t) \, dx,$$
  
$$I_k^- = a_k \int_{x_{k-1}}^{x_k} x \ln \frac{x}{x_{k-1}} q_k \, dx, \quad I_k^+ = a_{k+1} \int_{x_k}^{x_{k+1}} x \ln \frac{x_{k+1}}{x} q_{k+1} \, dx.$$

The finite difference scheme  $(N = 2, x_1 = h_1, x_2 = L = h_1 + h_2, \beta = h_2/h_1)$  is given in the following form

$$\begin{cases} \lambda_1(u_1 - u_0) = \gamma_1 J_2 + I_0, \\ \lambda_2 a_2(u_2 - u_1) = \gamma_2 R_1^+ + I_1^+ \gamma_1 J_1 + I_0^{(1)}, \\ f(u_2) - \lambda_2 a_2(u_2 - u_1) = \gamma_2 R_2^- + I_2^-, \end{cases}$$
(4.1)

where

$$J_{2} = \int_{0}^{h_{1}} x \ln \frac{h_{1}}{x} \dot{u}_{1}(x,t) dx, \quad I_{0} = \int_{0}^{h_{1}} x \ln \frac{h_{1}}{x} q_{1}(x,t) dx,$$
$$a_{2} = 1/\ln \frac{L}{h_{1}}, \quad J_{1} = \int_{0}^{h_{1}} x \dot{u}_{1}(x,t) dx, \quad I_{0}^{(1)} = \int_{0}^{h_{1}} x q_{1}(x,t) dx$$

In the stationary case, if  $q_k = const$ ,  $\dot{u}_k = 0$ , then

$$I_0 = \frac{q_1 h_1^2}{4}, \quad I_1^+ = q_2 \left( -0.5h_1^2 + 0.25(L^2 - h_1^2) \ln^{-1} \frac{L}{h_1} \right),$$
$$I_0^{(1)} = \frac{q_1 h_1^2}{2}, \quad I_2^- = q_2 \left( 0.5L^2 - 0.25(L^2 - h_1^2) \ln^{-1} \frac{L}{h_1} \right)$$

and the values  $u_0, u_1, u_2$  can be obtained from the system of equations

$$\begin{cases} \lambda_1(u_1 - u_0) = 0.25q_1h_1^2, \\ \lambda_2(u_2 - u_1) = 0.5(q_1 - q_2)h_1^2 \ln \frac{L}{h_1} + 0.25q_2(L^2 - h_1^2), \\ f(u_2) \ln \frac{L}{h_1} - \lambda_2(u_2 - u_1) = q_2 \left( 0.5L^2 \ln \frac{L}{h_1} - 0.25(L^2 - h_1^2) \right). \end{cases}$$
(4.2)

The exact solution of the stationary problem (1.1)–(1.3) satisfies expressions (4.2) and  $f(u_2) = 0.5(q_2L^2 + (q_1 - q_2)h_1^2)$ . This equation has a positive root if

$$\epsilon \sigma T_*^4 + \alpha_L T_L - 0.5 \left( q_2 (L^2 - h_1^2) + q_1 h_1^2 \right) > 0.$$
(4.3)

In the non-stationary case integrals  $J_1, J_2, R_1^+, R_2^-$  can be approximated by the following quadrature formulas

$$\frac{J_m}{h_1^2} = A_1^{(m)} V_1(0) + A_2^{(m)} V_1(1) + A_3^{(m)} V_1'(0) + B_1^{(m)} V_1''(0) + r_m, m = 1, 2, \quad (4.4)$$

$$\frac{J_m}{h_2^2} = A_1^{(m)} V(0) + A_2^{(m)} V(1) + A_3^{(m)} V'(1) + B_2^{(m)} \left( V''(1) + \frac{\beta V'(1)}{1+\beta} \right) + r_m, \quad (4.5)$$

where  $r_m = \frac{W^{(4)}(\xi_m)}{4!}C_m, \, \xi_m \in (0,1), \, m = 3,4; \, W = V_1, V,$ 

$$V(\bar{x}) = \dot{u}_2(h_1 + h_2\bar{x}, t), \quad V_1(\bar{x}) = u_1(h_1\bar{x}, t), \quad J_1 = h_1^2 \int_0^1 \bar{x} V_1(\bar{x}) \, d\bar{x},$$
  
$$J_2 = -h_1^2 \int_0^1 \bar{x} \ln(\bar{x}) V_1(\bar{x}) \, d\bar{x}, \quad J_3 = h_2^2 \int_0^1 (\beta^{-1} + \bar{x}) V(\bar{x}) \ln(1 + \beta \bar{x}) \, d\bar{x},$$
  
$$J_4 = -h_2^2 \int_0^1 (\beta^{-1} + \bar{x}) V(\bar{x}) \ln \frac{1 + \beta \bar{x}}{1 + \beta} \, d\bar{x}, \quad R_2^- = \frac{J_3}{\ln(1 + \beta)}, \quad R_1^+ = \frac{J_4}{\ln(1 + \beta)}$$

The unknown coefficients  $A_j^{(m)}, B_j^{(m)}$  can be determinated by using  $W(\bar{x}) = \bar{x}^i, i = 0, 1, 2, 3$ , and solving the system of linear algebraic equations with parameter  $\beta$ :

$$\begin{cases} g(i,m) = A_1^{(m)}0^i + A_2^{(m)} + iA_3^{(m)}0^{i-1} + i(i-1)B_2^{(m)}0^{i-2} \ (m=1,2), \\ g(i,m) = A_1^{(m)}0^i + A_2^{(m)} + iA_3^{(m)} + i(i-1+\beta/(1+\beta))B_2^{(m)} \ (m=3,4), \end{cases}$$

where

$$g(i,1) = 1/(i+2), \quad g(i,3) = \int_0^1 (\beta^{-1} + \bar{x}) \ln(1+\beta\bar{x})\bar{x}^i \, d\bar{x},$$
  
$$g(i,2) = 1/(i+2)^2, \quad g(i,4) = -\int_0^1 (\beta^{-1} + \bar{x}) \ln\frac{1+\beta\bar{x}}{1+\beta}\bar{x}^i \, d\bar{x}.$$

As an example, if  $\beta = 0.25$ , then we get

$$\begin{split} A_1^{(3)} &= 0.057368822, \quad A_2^{(3)} = 0.5789255389, \quad A_3^{(3)} = -0.1486490492, \\ B_2^{(3)} &= 0.02021926484, \quad C_3 = -0.01752470, \\ A_1^{(4)} &= 0.1505753322, \quad A_2^{(4)} = 0.2528510778, \quad A_3^{(4)} = -0.1135916341, \\ B_2^{(4)} &= 0.0202143207, \quad C_4 = -0.03082307. \end{split}$$

The other coeffi cients are given by

$$\begin{aligned} A_1^{(1)} &= \frac{3}{10}, \ A_2^{(1)} &= \frac{1}{5}, \ A_3^{(1)} &= \frac{2}{15}, \ B_1^{(1)} &= \frac{1}{40}, \ C_1 &= -\frac{1}{30}, \\ A_1^{(2)} &= \frac{21}{100}, \ A_2^{(2)} &= \frac{1}{25}, \ A_3^{(2)} &= \frac{16}{225}, \ B_1^{(2)} &= \frac{9}{800}, \ C_2 &= -\frac{11}{900} \end{aligned}$$

The following stiff system of three ODEs of the second order is obtained for finding  $u_0(t), u_1(t), u_2(t)$ :

$$\begin{cases} \gamma_{1}h_{1}^{2} \left[A_{1}^{(2)} \dot{u}_{0} + A_{2}^{(2)} \dot{u}_{1} + \frac{h_{1}^{2}}{2\lambda_{1}} B_{1}^{(2)} (\dot{q}_{1}(0,t) + \gamma_{1} \ddot{u}_{0})\right] + I_{0} = \lambda_{1}(u_{1} - u_{0}), \\ \frac{\gamma_{2}h_{2}^{2}}{\ln \frac{L}{h_{1}}} \left[A_{1}^{(4)} \dot{u}_{1} + A_{2}^{(4)} \dot{u}_{2} + \frac{h_{2}}{\lambda_{2}L} A_{3}^{(4)} f_{u}'(u_{2}) \dot{u}_{2} + \frac{h_{2}^{2}}{\lambda_{2}} B_{2}^{(4)} (\dot{q}_{2}(L,t) + \gamma_{2} \ddot{u}_{2})\right] \\ + \gamma_{1}h_{1}^{2} \left[A_{1}^{(1)} \dot{u}_{0} + A_{2}^{(1)} \dot{u}_{1} + \frac{h_{1}^{2}}{2\lambda_{1}} B_{1}^{(1)} (\dot{q}_{1}(0,t) + \gamma_{1} \ddot{u}_{0})\right] + I_{1}^{+} + I_{0}^{(1)} \\ = \lambda_{2} / \ln \frac{L}{h_{1}q} (u_{2} - u_{1}), \\ \frac{\gamma_{2}h_{2}^{2}}{\ln \frac{L}{h_{1}}} \left[A_{1}^{(3)} \dot{u}_{1} + A_{2}^{(3)} \dot{u}_{2} + \frac{h_{2}}{\lambda_{2}L} A_{3}^{(3)} f_{u}'(u_{2}) \dot{u}_{2} \frac{h_{2}^{2}}{\lambda_{2}} B_{2}^{(3)} (\dot{q}_{2}(L,t) + \gamma_{2} \ddot{u}_{2})\right] \\ + I_{2}^{-} = f(u_{2}) - \lambda_{2} / \ln \frac{L}{h_{1}} (u_{2} - u_{1}). \end{cases}$$

The initial conditions are

$$u_0(0) = \phi(0), \quad u_1(0) = \phi(h_1), \quad u_2(0) = \phi(L),$$
  
$$\dot{u}_0(0) = (2\lambda_1 \phi''(0) - q_1(0,0))/\gamma_1,$$
  
$$\dot{u}_2(0) = (\lambda_2(\phi''(L) + L^{-1}\phi'(L)) - q_2(L,0))/\gamma_2.$$

For the equation (1.6)  $(h_1 = h_2 = h = L/2, \alpha_L = \infty)$  the finite difference scheme (4.1) is defined as

$$u_1 - u_0 = J_2 + I_0, \quad T_L - u_1 = J_5 + I_1^+ + I_0^{(1)},$$
 (4.6)

where  $J_5 = \ln 2J_1 + J_0^*, \ J_0^* = \int_h^L x \ln \frac{L}{x} \dot{u}(x,t) \, dx, \ V(\bar{x}) = \dot{u}(\bar{x}h,t),$ 

$$\frac{J_5}{h^2} = \ln 2 \int_0^2 \bar{x} V(\bar{x}) \, d\bar{x} - \int_1^2 \bar{x} \ln \bar{x} V(\bar{x}) \, d\bar{x} = A_1^{(5)} V(0) + A_2^{(5)} V(1) + A_3^{(5)} V(2) + B_1^{(5)} V''(0) + \frac{B_2^{(5)}(\bar{x} V'(\bar{x}))'}{\bar{x}} \Big|_{\bar{x}=1} + \frac{V^{(5)}(\xi)}{5!} C_5, \ \xi \in (0,1).$$

If  $V(\bar{x}) = \bar{x}^i, i = \overline{0, 5}$ , then we get the following results

$$\begin{aligned} A_1^{(5)} &= \frac{98}{2475}, \ A_2^{(5)} &= \frac{6653}{9900}, \ A_3^{(5)} &= \frac{19}{495}, \\ B_1^{(5)} &= -\frac{53}{26400}, \ B_2^{(5)} &= \frac{287}{9900}, \ C_5 &= -\frac{4028}{121275} \end{aligned}$$

The following system of two ODEs of the second order is obtained for finding  $u_0, u_1$ :

$$\begin{cases} h^2 (A_1^{(2)} \dot{u}_0 + A_2^{(2)} \dot{u}_1) + 0.5 h^4 B_1^{(2)} (\dot{q}_0 + \ddot{u}_0) + I_0 = u_1 - u_0, \\ h^2 (A_1^{(5)} \dot{u}_0 + A_2^{(5)} \dot{u}_1 + A_3^{(5)} \dot{u}_2) + h^4 (0.5 B_1^{(5)} (\dot{q}_0 + \ddot{u}_0) + B_2^{(5)} (\dot{q}_1 + \ddot{u}_1)) \\ + I_1^+ + I_0^{(1)} = T_L - u_1. \end{cases}$$

*Example 2.* [5] If q = 0,  $T_L = 0$ ,  $\dot{u}_2 = 0$ ,  $\phi(x) = J_0(\mu x/L)$  is the Bessel's function of first kind,  $\mu = 2.404825558$  the first positive root of equation  $\mathcal{J}(\mu) = 0$ , then the exact solution is  $u(x,t) = \exp(-(\mu/L)^2 t) J_0(\mu x/L)$  with

$$u_0(t) = \exp\left(-\left(\frac{\mu}{L}\right)^2 t\right), u_1(t) = J_0\left(\frac{\mu}{2}\right) \exp\left(-\left(\frac{\mu}{L}\right)^2 t\right), J_0\left(\frac{\mu}{2}\right) = 0.6699297389.$$

The results of calculations are presented in Table 2, where L = 2,  $u_{0*}$ ,  $u_{1*}$  are the exact values,  $u_{0p}$ ,  $u_{1p}$  approximate values.

t	$u_{0*}$	$u_{0p}$	$u_{1*}$	$u_{1p}$
.1	.86538	.86540	.57974	.57972
.2	.74889	.74891	.50171	.50165
.5	.48534	.48530	.32515	.32505
.9	.27220	.27212	.18236	.18226

**Table 2.** The values u(0, t), u(1, t) in the time t.

*Example 3.* [1] For heat transfer in cylindrical wire-metal (copper) conductor ( $x \in [0, h_1]$ ) with insulation ( $x \in (h_1, L]$ ) the term  $q_1$  describes Joule's heat generation

and it can be written as [1]  $q_1 = -\sigma_0^{-1} \frac{I^2}{S^2}$ , where  $\sigma_0[\frac{1}{\Omega.m}]$  is the electric conductivity of metal, I[A] is the electric current,  $S[m^2]$  is the cross-section of the wire  $(S = \pi h_1^2)$ . The numerical results are obtained for the following values of parameters:

$$\begin{split} h_1 &= 4 \ [mm], \ \ L = 5 \ [mm], \ \ \beta = 0.25, \ \ I = 173 \ [A], \ \ \alpha_L = 1 \ \left[\frac{W}{m^3 K}\right], \\ \epsilon &= 0.5, \ \ \sigma_0^{-1} = 1.7 \cdot 10^{-8} \ [\Omega m], \ \ c_1 = 410 \ \left[\frac{J}{kgK}\right], \ \ c_2 = 840 \ \left[\frac{J}{kgK}\right], \\ \rho_1 &= 8960 \ \left[\frac{kg}{m^3}\right], \ \ \rho_2 = 500 \ \left[\frac{kg}{m^3}\right], \ \ q_1 = -0.217 \cdot 10^7 \ \left[\frac{W}{m^3}\right], \ \ q_2 = 0, \\ \lambda_1 &= 400 \ \left[\frac{W}{mK}\right], \ \ \lambda_2 = 0.2 \ \left[\frac{W}{mK}\right], \ \ T_* = T_L = 293 \ [K]. \end{split}$$

We have calculated stationary and non-stationary solutions in two cases:

1) with radiation  $u_0 = 316.78[K], u_1 = 316.76[K], u_2 = 297.42[K],$ 

2) without radiation  $u_0 = 329.69[K]$ ,  $u_1 = 329.67[K]$ ,  $u_2 = 310.33[K]$ .

In the non-stationary case the results are presented in tables 3 (with radiation) and 4 (without radiation). We can see the effect of radiation.

**Table 3.** The values  $u_0 = u_1, u_2, \dot{u}_0, \dot{u}_2$  in time t with radiation.

t	$u_0$	$\dot{u}_0$	$u_2$	$\dot{u}_2$
1	293.6	0.560	293.1	0.094
10	298.1	0.450	293.9	0.086
50	309.7	0.171	296.1	0.032
100	314.7	0.051	297.0	0.009
150	316.2	0.015	297.3	0.003
200	316.6	0.004	297.4	0.001

**Table 4.** The values  $u_0 = u_1, u_2, \dot{u}_0, \dot{u}_2$  in time t without radiation.

t	$u_0$	$\dot{u}_0$	$u_2$	$\dot{u}_2$
1	293.6	0.560	293.1	0.193
10	298.3	0.490	295.3	0.240
50	312.8	0.260	302.3	0.132
100	322.0	0.120	306.6	0.061
150	326.1	0.055	308.6	0.026
200	328.0	0.025	309.5	0.012

# 5. The Spherical Coordinates



**Figure 3.** The calculated temperatures  $u_0$  and  $u_1$  in the points x = 0 and  $x = x_1$  in the case of spherical coordinates.

In the spherical coordinates (see Fig. 3) we get the following coeffi cients

$$p(x) = x^{2}, \quad R_{k}^{-} = \frac{a_{k}}{x_{k-1}} \int_{x_{k-1}}^{x_{k}} x(x - x_{k-1})\dot{u}_{k}(x, t) \, dx,$$
$$a_{k} = \left(\frac{1}{x_{k-1}} - \frac{1}{x_{k}}\right)^{-1}, \quad R_{k}^{+} = \frac{a_{k+1}}{x_{k+1}} \int_{x_{k}}^{x_{k+1}} x(x_{k+1} - x)\dot{u}_{k+1}(x, t) \, dx.$$

The finite-difference scheme (N = 2) is given by (4.1), where

$$J_{2} = h_{1}^{-1} \int_{0}^{h_{1}} x(h_{1} - x)\dot{u}_{1}(x, t) dx, \quad I_{0} = h_{1}^{-1} \int_{0}^{h_{1}} x(h_{1} - x)q_{1} dx,$$
$$J_{1} = \int_{0}^{h_{1}} x^{2}\dot{u}_{1}(x, t) dx, \quad I_{0}^{(1)} = \int_{0}^{h_{1}} x^{2}q_{1} dx, \quad a_{2} = \frac{L}{\beta}, \ \beta = \frac{h_{2}}{h_{1}}.$$

In the stationary case  $(q_k = const)$  we get:

$$I_0 = \frac{q_1 h_1^2}{6}, \quad I_0^{(1)} = \frac{q_1 h_1^3}{3},$$
  

$$I_2^- = \frac{q_2 L}{6\beta} (2\beta L^2 + h_1^2 - L^2), \quad I_1^+ = \frac{q_2}{6\beta} (L^3 - h_1^3 - 3\beta h_1^3).$$

From (4.1) it follows that

$$\begin{cases} \lambda_1(u_1 - u_0) = \frac{q_1 h_1^2}{6}, \\ \frac{\lambda_2 \beta}{L} (u_2 - u_1) = \frac{q_1 h_1^3}{3} + \frac{q_2}{6\beta} (L^3 - h_1^3 - 3\beta h_1^3), \\ f(u_2) - \frac{\lambda_2 \beta}{L} (u_2 - u_1) = \frac{q_2 L}{6\beta} (2\beta L^2 + h_1^2 - L^2). \end{cases}$$
(5.1)

The exact solution of the problem (1.1)–(1.3) satisfies (5.1) and

$$f(u_2) - \frac{q_2 L^3}{3} = \frac{h_1^3}{3} (q_1 - q_2).$$

This equation has a positive root if

$$\epsilon \sigma T_*^4 + \alpha_L T_L - \frac{1}{3} \left( q_2 (L^3 - h_1^3) + q_1 h_1^3 \right) > 0.$$

In the non-stationary case we have the following quadrature formulas  $(x = \bar{x}h_1, x = h_1 + \bar{x}h_2)$ :

$$\frac{J_m}{h_1^3} = A_1^{(m)} V_1(0) + A_2^{(m)} V_1(1) + A_3^{(m)} V_1'(0) + B_1^{(m)} V_1''(0) + r_m, \ m = 1, 2,$$
  
$$\frac{J_m}{h_2^2 L} = A_1^{(m)} V(0) + A_2^{(m)} V(1) + A_3^{(m)} V'(1) + B_2^{(m)} \left( V''(1) + \frac{2\beta V'(1)}{1+\beta} \right) + r_m,$$

where  $r_m = \frac{W^{(4)}(\xi_m)}{4!}C_m, \xi_m \in (0,1), m = 3,4; W = V, V_1, m = 1,2,3,4,$ 

$$J_{1} = h_{1}^{3} \int_{0}^{1} \bar{x}^{2} V_{1}(\bar{x}) \, d\bar{x}, \ J_{2} = h_{1}^{3} \int_{0}^{1} \bar{x}(1-\bar{x}) V_{1}(\bar{x}) \, d\bar{x}, \ J_{3} = \beta R_{2}^{-}, \ J_{4} = \beta R_{1}^{+},$$
  
$$J_{3} = h_{2}^{2} L \int_{0}^{1} (\beta \bar{x} + 1) V(\bar{x}) \, d\bar{x}, \ J_{4} = \frac{h_{2}^{2} L}{1+\beta} \int_{0}^{1} (\beta \bar{x} + 1)(1-\bar{x}) V(\bar{x}) \, d\bar{x}.$$

The undefinite coeffi cients are given by:

$$\begin{split} A_1^{(1)} &= A_2^{(1)} = \frac{1}{6}, \ A_3^{(1)} = \frac{1}{12}, \\ B_1^{(1)} &= \frac{1}{60}, \ C_1 = -\frac{1}{42}, \ A_1^{(2)} = \frac{2}{15}, \\ A_2^{(2)} &= \frac{1}{30}, \ A_3^{(2)} = \frac{1}{20}, \ B_1^{(2)} = \frac{1}{120}, \ C_2 = -\frac{1}{105}, \ A_1^{(3)} = \frac{\beta + 3}{60}, \\ A_2^{(3)} &= \frac{19\beta + 27}{60}, \ A_3^{(3)} = -\frac{5\beta^2 + 13\beta + 7}{60(1 + \beta)}, \ B_2^{(3)} = \frac{\beta + 2}{120}, \ C_3 = -\frac{\beta + 2}{120}, \\ A_1^{(4)} &= \frac{\beta + 6}{30(1 + \beta)}, \ A_2^{(4)} = \frac{4\beta + 9}{30(\beta + 1)}, \ A_3^{(4)} = -\frac{2\beta^2 + 7\beta + 4}{30(1 + \beta)^2}, \\ B_2^{(4)} &= \frac{\beta + 3}{120(1 + \beta)}, \ C_4 = -\frac{2\beta + 7}{210(1 + \beta)}. \end{split}$$

Therefore we obtain the following system of ODEs:

$$\begin{cases} \gamma_{1}h_{1}^{2} \left[A_{1}^{(2)} \dot{u}_{0} + A_{2}^{(2)} \dot{u}_{1} + \frac{h_{1}^{2}B_{1}^{(2)}}{3\lambda_{1}} \left(\dot{q}_{1}(0,t) + \gamma_{1} \ddot{u}_{0}\right)\right] + I_{0} = \lambda_{1}(u_{1} - u_{0}), \\ \frac{\gamma_{2}h_{2}^{2}L}{\beta} \left[A_{1}^{(4)} \dot{u}_{1} + A_{2}^{(4)} \dot{u}_{2} + \frac{h_{2}}{\lambda_{2}L^{2}} A_{3}^{(4)} f_{u}'(u_{2}) \dot{u}_{2} + \frac{h_{2}^{2}}{\lambda_{2}} B_{2}^{(4)} \left(\dot{q}_{2}(L,t) + \gamma_{2} \ddot{u}_{2}\right)\right] \\ + \gamma_{1}h_{1}^{3} \left[A_{1}^{(1)} \dot{u}_{0} + A_{2}^{(1)} \dot{u}_{1} + \frac{h_{1}^{2}}{3\lambda_{1}} B_{1}^{(1)} \left(\dot{q}_{1}(0,t) + \gamma_{1} \ddot{u}_{0}\right)\right] \\ + I_{1}^{+} + I_{0}^{(1)} = \frac{\lambda_{2}L}{\beta} \left(u_{2} - u_{1}\right), \\ \frac{\gamma_{2}h_{2}^{2}L}{\beta} \left[A_{1}^{(3)} \dot{u}_{1} + A_{2}^{(3)} \dot{u}_{2} + \frac{h_{2}}{\lambda_{2}L^{2}} A_{3}^{(3)} f_{u}'(u_{2}) \dot{u}_{2} + \frac{h_{2}^{2}}{\lambda_{2}} B_{2}^{(3)} \left(\dot{q}_{2}(L,t) + \gamma_{2} \ddot{u}_{2}\right)\right] \\ + I_{2}^{-} = f(u_{2}) - \frac{\lambda_{2}L}{\beta} \left(u_{2} - u_{1}\right). \end{cases}$$

The initial conditions are given by

$$\begin{cases} u_0(0) = \phi(0), \ u_1(0) = \phi(h_1), \ u_2(0) = \phi(L), \\ \dot{u}_0(0) = (3\lambda_1\phi''(0) - q_1(0,0))/\gamma_1, \\ \dot{u}_2(0) = \left(\lambda_2(\phi''(L) + 2L^{-1}\phi'(L)) - q_2(L,0)\right)/\gamma_2. \end{cases}$$

For the equation (1.6)  $(h_1 = h_2 = h = L/2, \alpha_L = \infty, \dot{u}_2 = 0)$  we have the fi nite difference scheme in the form

$$u_1 - u_0 = J_2 + I_0, \quad T_L - u_1 = J_5 + I_1^+ + I_0^{(1)},$$
 (5.2)

where

$$\begin{split} J_5 &= \frac{J_1 + J_0^*}{L}, \ J_0^* = \int_h^L x(L - x)\dot{u}(x,t) \, dx, \\ \frac{J_5}{h^2} &= \frac{1}{2} \Big( \int_0^1 \bar{x}^2 V(\bar{x}) \, d\bar{x} + \int_1^2 \bar{x}(2 - \bar{x}) V(\bar{x}) \, d\bar{x} \Big) = A_1^{(5)} V(0) + A_2^{(5)} V(1) \\ &+ A_3^{(5)} V(2) + B_1^{(5)} V''(0) + \frac{B_2^{(5)}(\bar{x}^2 V'(\bar{x}))'}{\bar{x}^2} \Big|_{\bar{x}=1} + \frac{B_3^{(5)}(\bar{x}^2 V'(\bar{x}))'}{\bar{x}^2} \Big|_{\bar{x}=2} \\ &+ \frac{V^{(6)}(\xi)}{6!} C_5, \ \xi \in (0,2), \quad V(\bar{x}) = \dot{u}(\bar{x}h,t) = \bar{x}^i, \ i = 0, \dots, 6, \\ A_1^{(5)} &= \frac{1}{1008}, \ A_2^{(5)} = \frac{115}{252}, \ A_3^{(5)} = \frac{43}{1008}, \ B_1^{(5)} = \frac{1}{5040}, \\ B_2^{(5)} &= \frac{107}{5040}, \ B_3^{(5)} = -\frac{1}{1260}, \ C_5 = -\frac{1}{315}. \end{split}$$

If we add in expression  $J_5/h^2$  the term  $A_4^{(5)}V'(0)$  with error term  $\frac{V^{(7)}(\xi)}{7!}C_5$ , then we have the following values of coefficients:

$$A_{1}^{(5)} = B_{1}^{(5)} = 0, \quad A_{2}^{(5)} = \frac{115}{252}, \quad A_{3}^{(5)} = \frac{11}{252}, \quad B_{2}^{(5)} = \frac{313}{15120},$$
  

$$B_{3}^{(5)} = -B_{2}^{(5)}, \quad A_{4}^{(5)} = -B_{2}^{(5)}, \quad C_{5} = -\frac{59}{3780}.$$
(5.3)

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$$h^{2}(A_{1}^{(2)}\dot{u}_{0} + A_{2}^{(2)}\dot{u}_{1}) + h^{4}\left(B_{1}^{(2)}\frac{\dot{q}_{0} + \ddot{u}_{u}}{3} + B_{2}^{(2)}(\dot{q}_{1} + \ddot{u}_{1})\right) + I_{0} = u_{1} - u_{0},$$
(5.4)

$$h^{2}(A_{1}^{(5)}\dot{u}_{0} + A_{2}^{(5)}\dot{u}_{1}) + h^{4}\left(B_{1}^{(5)}\frac{\dot{q}_{0} + \ddot{u}_{0}}{3} + B_{2}^{(5)}(\dot{q}_{1} + \ddot{u}_{1}) + B_{3}^{(5)}\dot{q}_{2}\right)$$
(5.5)  
+  $I_{1}^{+} + I_{0}^{(1)} = T_{L} - u_{1}.$ 

*Example 4.* If q = 0,  $T_L = 0$ ,  $\phi(x) = x^{-1} \sin(\pi x/L)$ , then the exact solution is

$$u(x,t) = \frac{1}{x} \exp(-(\pi/L)^2 t) \sin(\pi x/L),$$
  
$$u_0(t) = \frac{\pi}{L} \exp(-(\pi/L)^2 t), \quad u_1(t) = \frac{2}{L} \exp(-(\pi/L)^2 t).$$

We have solved the system (5.4)–(5.5) with initial conditions

$$u_0(0) = \frac{\pi}{L}, \ u_1(0) = \frac{2}{L}, \ \dot{u}_1(0) = -\frac{2\pi^2}{L^3}, \ \dot{u}_0 = -\left(\frac{\pi}{L}\right)^3, \ L = 2.$$

The results are presented in Table 5.

**Table 5.** The values u(0, t), u(1, t) in the time t.

t	$u_{0*}$	$u_{0p}$	$u_{1*}$	$u_{1p}$
.1	1.22733	1.22741	0.781344	0.781339
.2	0.95897	0.95907	0.610498	0.610488
.5	0.45744	0.45749	0.291213	0.291198
.9	0.17049	0.17050	0.108537	0.108527

### 6. Conclusions

- 1. The proposed method allows us to reduce 1D heat transfer problem in Cartesian, cylindrical and spherical coordinates to the system of the ordinary differential equations of the second order.
- The described methods make it possible to find the distribution of temperature in the case of different layers with the heat source in between the layers and on layers borders.
- 3. In different coordinates it is possible to enlarge the accuracy of the given algorithm, when second order derivatives are used instead of fi rst order derivatives.
- 4. Such formulations have a big practical importance as compared to Cartesian coordinates, e.g. for analysis of heat transfer in cylindrical wire-metal (coper) conductor with insulation.

#### References

- A. Ilgevičus and H.-D. Liess. Calculation of the heat transfer in cylindrical wires and electric fuses by implicit finite volume method. *Mathematical Modelling and Analysis*, 8(3), 217 – 228, 2003.
- [2] H. Kalis. Effective finite-difference schemes for solving some heat transfer problems with convection in multilayer media. *Int. Journ. of Heat and Mass Transfer*, 43, 4467 – 4474, 2000.
- [3] H. Kalis and I. Kangro. Simple algorithm's for the calculation of heat transport problem in plate. *Mathematical Modelling and Analysis*, 6(1), 85 – 96, 2001.
- [4] H. Kalis and I. Kangro. Simple methods of engineering calculation for solving heat transfer problems. *Mathematical Modelling and Analysis*, **8**(1), 33 42, 2003.
- [5] H. Kalis and A. Lasis. Simple algorithm's for calculation the axial-symmetric heat transport problem in a cylinder. *Mathematical Modelling and Analysis*, 6(2), 262 – 269, 2001.

#### Kai kurių inžinerinių, dvisluoksnėje srityje, šilumos laidumo uždavinių tikslumo didinimas

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Šiame straipsnyje yra nagrinėjami paprasti dvisluoksnės srities šilumos laidumo problemos modeliavimo algoritmai, keičiant diferencialines lygtis dalinėmis išvestinėmis į paprastas diferencialines lygtis. Parodoma, kad didesnio tikslumo pasiekimui, vietoje pirmos eilės paprastų diferencialinių lygčių pradinio uždavinio nagrinėjamos antros eilės diferencialinės lygtys. Ši proced<sup>-</sup>ura leidžia gauti paprastą inžinerinį dvisluoks**t**es srities šilumos laidumo lygties sprendinį stačiakampėje, cilindrinėje (su ašių simetrija) ir sferinėje (su spinduline simetrija) koordinačių sistemoje. Tiksli baigtinių skirtumų schema buvo sudaryta stacionariam atvejui.